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# Understanding and predicting viscous, elastic, plastic flows

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**Abstract.** Foams, gels, emulsions, polymer solutions, pastes [1] and even cell assemblies [2,3] display both liquid and solid mechanical properties. On a local scale, such “soft glassy” systems [4] are disordered assemblies of deformable rearranging units, the complexity of which gives rise to their striking flow behaviour [5]. On a global scale, experiments show that their mechanical behaviour depends on the orientation of their elastic deformation with respect to the flow direction [6], thus requiring a description by tensorial equations for continuous materials. However, due to their strong non-linearities, the numerous candidate models [7–10] have not yet been solved in a general multi-dimensional geometry to provide stringent tests of their validity. We compute the first solutions of a continuous model [11] for a discriminant benchmark, namely the flow around an obstacle [12,13]. We compare it with experiments of a foam flow [14] and find an excellent agreement with the spatial distribution of all important features: we accurately predict the experimental fields of velocity, elastic deformation, and plastic deformation rate in terms of magnitude, direction, and anisotropy. We analyse the role of each parameter, and demonstrate that the yield strain is the main dimensionless parameter required to characterize the materials. We evidence the dominant effect of elasticity, which explains why the stress does not depend simply on the shear rate [15–18]. Our results demonstrate that the behaviour of soft glassy materials cannot be reduced to an intermediate between that of a solid and that of a liquid: the viscous, the elastic and the plastic contributions to the flow, as well as their couplings, must be treated simultaneously. Our approach opens the way to the realistic multi-dimensional prediction of complex flows encountered in geophysical, industrial and biological applications, and to the understanding of the link between structure and rheology of soft glassy systems.

**PACS** – 47.57.Bc Foams and emulsions – 83.60.La Viscoplasticity; yield stress – 83.60.Df Nonlinear viscoelasticity – 02.60.Cb Numerical simulation; solution of equations

## 1 Introduction

Materials such as pastes or polymer solutions display both solid-like and liquid-like behaviors; they are successfully described by visco-elastic (VE) or visco-plastic (VP) models. However, we still lack testable predictions of the time- and space-dependent flows of soft glassy materials [6, 7], that are made of disordered assemblies of deformable, rearranging units [6, 19]. It had been suggested that fluctuations remain relevant even at large scale, in which case detailed statistical theories of long-range correlations and avalanches would be required [20, 21]. This view is challenged by recent experiments which suggest that even these materials can be treated as continuous materials described by tensorial equations [6, 14]; thus in principle partial differential equations could lead to the long-awaited predictions.

Based on our experience with foams, we believe that the reason of the difficulty comes from the fact that these materials are simultaneously viscous, elastic, and plastic (VEP). Under small deformation, a foam reversibly comes

back to its shape; at large deformation, it can be irreversibly sculpted and gets a new shape; under an increasing deformation rate, it irreversibly flows, with an increasing viscous stress [22–26]. Existing continuous models of foam flows include for instance either a phenomenological scalar description [17, 27], or a complete tensorial description of the elasticity [9] or plasticity [19] based on the micro-structure. Overall, continuous VEP models tend to successfully reproduce some experimental measurements, such as elastic and loss moduli, or compliance (see for instance refs. [3, 9, 11, 25]).

We want here to understand and predict VEP flows. Our approach is to test whether a continuous VEP model can capture the essence and complexity of the flow properties, especially the elasticity and its advection by the flow. We thus need a VEP model as simple as possible: with linear and isotropic coefficients, without fluctuations nor long-range interactions. We require to use only physically relevant parameters, which in principle are measurable. We need a tensorial model in a multidimensional space (what follows applies both in 2D and 3D). We need a closed system of equations: constitutive equations, specific to the material under consideration; generic conservation laws, in the spirit of hydrodynamics; and closure equations, to unify the solid and liquid descriptions.

To really test a model, and also lead to practical applications, we address the full spatial and orientational heterogeneity of a flow. We need to investigate a controlled, reproducible flow which displays significantly different  $V$ ,  $E$  and  $P$  contributions. It should involve a large range of

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shear rates, of tensorial orientations, and of elastic deformations. It should display elastic deformations either parallel, perpendicular or at odd angles with respect to the shear rate. Most discriminant, it should actually depends on two or more dimensions of space, so that the advection of elastic stress couples the shear and normal stress components (see eqs. (8-9)). This latter point, overlooked by most models, is crucial to test whether the elastic stress could be entirely determined by the shear rate, leading to a VP rheology (such as Bingham [28] or Herschel-Bulkley [29]). Recent experiments [15,16] suggest that it is not the case: in fact, the stress, the shear rate and the elastic deformation should be treated as independent variables, so that a full VEP treatment is required.

One such flow is the well documented flow around an obstacle [12] (Fig. 2). It displays a strong spatial heterogeneity, simultaneous VEP behaviors, a large range of elastic deformations, several elongation and rotation rates, and various relative orientations of the relevant tensors [19]. It enables to follow a bubble at different stages, while it stretches, then while it relaxes: thus, even in a steady flow, transient effects and relaxation times are apparent. It is classically used as a stringent test to discriminate between different models [30] (Fig. 3).

We use foams as model systems of VEP materials. Experiments with foams or emulsions, especially in 2 dimensions, enable an easy, simultaneous visualisation of the micro-structure (bubbles or droplets, which act as tracers of velocity and deformation) and the large scale (global flow heterogeneities).

To compare measurements from discrete experiments with continuous predictions from partial differential equations, we use the experimental tools we have developed [19]. V, E and P contributions are expressed in the same units, favoring a unified description of solid and liquid behaviors; each of them is valid in all regimes (so that e.g. elastic contributions can be measured even out of the elastic regime). These are local (in situ) measurements which link the foam structure and rheology.

Our plan is as follows. In order to make this paper self-contained, Section 2 recalls Saramito model [11]; here we write it by emphasizing how it unifies solid and liquid-like behaviours. Section 3 briefly recalls the experimental methods. It explains the careful resolution algorithm we had to develop to deal with the strong non-linearities of VEP equations, and a well controlled space- and time discretisation. Section 4 first fits the solutions to a wet foam flow, fixing the values of the parameters. It then turns to predictions of a foam with a higher yield strain, which accurately match a dry foam flow. Section 5 includes concluding remarks. An Appendix explains why it is important to solve the full time-dependent equations, even if we are interested only in stationary flows. Another Appendix lists part of our systematic tests of the predictions: all of them are available upon request.

## 2 Model

### 2.1 Characteristics and past results of the model

We choose to use Saramito's VEP model [11] because it has a positive dissipation, thanks to the convexity of its energy function. This implies that, at least for small deformations, it obeys by construction the second principle of thermodynamics. It includes as limiting cases both the VP Bingham model [28] and the VE Oldroyd model [31], and generalises them to VEP. It can extend to large deformations and high velocities.

It has already been used for space- and time-dependent predictions [11]. It has first been used to solve simple cases, such as steady uniaxial elongation. It has also applied to oscillatory regimes, and calculations of  $G'$ ,  $G''$ , rigidity and loss moduli.

It has then been implemented to calculate both time-dependent and steady shear Couette flows, which depend on one space variable (circular [32] or planar [33] geometries). It involves two strong non-linearities, intrinsic to VEP flows, and thus independent of the model: one because the plasticity appears above a yield point (eq. 3), and the other because of the advection of elastic stress (eqs. (8-9)). Despite these unavoidable difficulties, the model has been solved [34,35]. The resulting velocity, elasticity and plasticity fields agreed with experimental measurements.

This improved our understanding of Couette flows [35]. The localization of the velocity field results from the stress heterogeneity, so that the circular geometry by itself can

induce localisation. In planar geometry, where the stress is a priori homogeneous, localisation necessarily relies on another cause of heterogeneity, such as an external friction. Initial normal stresses can be preserved even in a steady flow, so that there are residual normal stresses which depend on initial conditions linked with the foam preparation method: the steady flow is not unique. Despite its simplicity, a Couette flow displays specific VEP features [34]. For instance, at the boundary of the localised region, the discontinuity of velocity gradient depends on the residual normal stresses, and thus on initial conditions. For all these reasons, and because the range of experimental data is limited, Couette flows have only a limited ability to discriminate between models, or between parameter values.

### 2.2 Notations and equations

#### 2.2.1 Constitutive equations: solid mechanics

We start with constitutive equations specific of a semi-fluid semi-solid material (see Fig. 1). In order to emphasize the dominant role of elasticity, we express them here in terms of deformations and their rates, as is usual in the context of solid mechanics:

$$\sigma_{tot} = -pI + 2\eta_1\dot{\varepsilon} + 2\mu\varepsilon^e, \quad (1)$$

$$\dot{\varepsilon} = \dot{\varepsilon}^e + \dot{\varepsilon}^p, \quad (2)$$

$$\dot{\varepsilon}^p = \begin{cases} \frac{1}{\lambda} \frac{|\varepsilon^e| - \varepsilon_Y}{|\varepsilon^e|} \varepsilon^e & \text{when } |\varepsilon^e| > \varepsilon_Y, \\ 0 & \text{otherwise.} \end{cases} \quad (3)$$

Eq. (1) is a constitutive equation for the Cauchy stress  $\sigma_{tot}$ , with pressure, viscous and elastic terms. Here  $p$  is the

pressure,  $I$  the identity tensor;  $\dot{\varepsilon}$  is the total deformation rate tensor,  $\eta_1$  is the viscosity of the material apparent at small deformation (in the foam, it includes the dissipation inside the soap films);  $\varepsilon^e$  is the elastic deformation tensor,  $\mu$  the shear modulus.

Eq. (2) recalls that  $\dot{\varepsilon}$  is shared between elastic  $\dot{\varepsilon}^e$  and plastic  $\dot{\varepsilon}^p$  contributions.

Eq. (3) is a plasticity equation, which specifies  $\dot{\varepsilon}^p$  by stating that plasticity increases above the yield strain  $\varepsilon_Y$ : under a strong shear rate, the elastic deformation  $|\varepsilon^e|$  can become higher than  $\varepsilon_Y$ . Here  $|\varepsilon^e|$  is the norm of  $\varepsilon^e$ , which we define as  $|\varepsilon^e| = [(\varepsilon_{yx}^e)^2 + (\varepsilon_{xx}^e - \varepsilon_{yy}^e)^2/4]^{1/2}$  to facilitate the comparison with a scalar shear (fixed shear direction  $xy$  with a given amplitude  $\varepsilon$ ), *i.e.*  $\varepsilon_{xx}^e - \varepsilon_{yy}^e = 0$  and  $\varepsilon_{yx}^e = \varepsilon$ , so that  $|\varepsilon^e| = \varepsilon$  [6]; the externally measured scalar shear is then  $\gamma = 2|\varepsilon^e|$ . Another acceptable definition would be the euclidian norm of the deviatoric elastic strain tensor [35],  $[(\varepsilon_{xx}^e - \varepsilon_{yy}^e)/2]^2 + (\varepsilon_{xy}^e)^2 + (\varepsilon_{yx}^e)^2 + [(\varepsilon_{yy}^e - \varepsilon_{xx}^e)/2]^2]^{1/2} = [2(\varepsilon_{yx}^e)^2 + (\varepsilon_{xx}^e - \varepsilon_{yy}^e)^2/2]^{1/2}$ , which is  $\sqrt{2}$  times larger than  $|\varepsilon^e|$ . The value of  $\varepsilon_Y$  is defined consistently with that of  $|\varepsilon^e|$ .

In soft disordered materials, plasticity is related with local rearrangements. In foams, these happen when bubbles swap neighbors and are called “T1” processes [22, 23, 36]. They create a transient local deformation. Here  $\lambda$  is the relaxation time of the material after such a local deformation [37]. We can construct the dissipation due to plasticity, which has the dimension of an effective viscosity  $\eta_2 = \lambda\mu$ : it determines the loss modulus at large amplitude. The softness and deformability of the mate-

rial appears in the value of  $\varepsilon_Y$ , of order of unity, so that both the elastic and plastic behaviors are experimentally observable; its glassy (*i.e.* disordered) nature implies that  $\lambda$ ,  $\mu$ ,  $\varepsilon_Y$  are isotropic [6].

## 2.2.2 Conservation equations: fluid mechanics

Generic conservation equations for an isothermal flow are expressed in terms of the velocity  $\mathbf{v}$  and its derivatives, as is usual in the context of fluid mechanics:

$$\rho \dot{\mathbf{v}} = \mathbf{div} \sigma_{tot} + \mathbf{f}_{ext}, \quad (4)$$

$$\mathbf{div} \mathbf{v} = 0. \quad (5)$$

Eq. (4) is the equation of dynamics;  $\rho$  is the density,  $\dot{\mathbf{v}} = \partial_t \mathbf{v} + \mathbf{v} \cdot \nabla \mathbf{v}$ , and  $\rho \dot{\mathbf{v}}$  denotes the inertia term (which we neglect below, see Section 3.3);  $\mathbf{f}_{ext}$  is the external force: in the bidimensional flow experiments studied below, the friction on the top and bottom boundaries (*e.g.* horizontal glass or perspex plates) is approximately  $\mathbf{f}_{ext} = -k\mathbf{v}$ , where  $k$  is a constant. We find experimentally (see Section 3.3) that  $k$  is small enough that in the present flows the effect of  $\mathbf{f}_{ext}$  is not measurable, so that we neglect it.

Eq. (5) describes the incompressibility of the flow: it applies to slow flows when the compressibility modulus is much higher than the shear modulus, as is the case in foams [22–24].

## 2.2.3 Closing equations: linking solid and fluid mechanics

To close the system of eqs. (1-3) and eqs. (4-5) requires to couple the deformations to the velocity.

First, the total deformation rate equals the symmetrized velocity gradient:

$$\dot{\varepsilon} = \frac{\nabla \mathbf{v} + \nabla \mathbf{v}^T}{2}. \quad (6)$$

Second, the time variation of the elastic deformation tensor  $\varepsilon^e$  accounts for its advection by the flow velocity  $\mathbf{v}$ . The model should be objective, that is, the expression of the equations should remain the same for an observer who has a movement of translation or rotation with respect to the experiment. The advection of the elastic deformation tensor is thus described with a frame invariant tensorial derivative [38]:

$$\dot{\varepsilon}^e = \frac{\mathcal{D}\varepsilon^e}{Dt}. \quad (7)$$

The objective derivative is [38, 11]:

$$\frac{\mathcal{D}\varepsilon^e}{Dt} = \frac{\partial \varepsilon^e}{\partial t} + (\mathbf{v} \cdot \nabla) \varepsilon^e + \beta_a(\varepsilon^e, \nabla \mathbf{v}), \quad (8)$$

where

$$\beta_a(\varepsilon^e, \nabla \mathbf{v}) = \varepsilon^e \cdot W(\mathbf{v}) - W(\mathbf{v}) \cdot \varepsilon^e - a(D(\mathbf{v}) \cdot \varepsilon^e + \varepsilon^e \cdot D(\mathbf{v})), \quad (9)$$

Here  $W(\mathbf{v}) = 0.5(\nabla \mathbf{v} - \nabla \mathbf{v}^T)$  is the antisymmetric part of the velocity gradient, and  $a \in [-1, 1]$  is the so-called “ $a$  parameter” [11, 38], which effect is discussed in section 3.3.

## 3 Methods

### 3.1 Experimental methods

Experimental set-ups have been described in refs. [14, 19, 39]. Bubble distributions are monodisperse in size (area  $16 \text{ mm}^2$ ) and disordered in geometry (shape, number of

neighbors). While bubbles pass through the field of view, no rupture is observed, and coarsening is negligible.

The wet foam is prepared by direct bubbling into the 1 m long channel. At its entrance in the channel, it displays normal differences in elastic deformation,  $\varepsilon_{xx}^e - \varepsilon_{yy}^e$  (axis 2 in Fig. 7).

For the dry foam, bubbles pass first through a chamber (in which the foam drains): this chamber enables to vary the liquid fraction over more than three decades, and homogeneizes the foam while relaxing its normal differences in elastic deformation [39].

Experimental measurements treat solid and liquid behaviors with a unique set of mutually compatible tools [19]. We derive the continuous description directly from averages over almost a thousand of images of discrete measurements performed on all bubbles which can be automatically identified using image analysis, that is, which do not touch the obstacle. The entrance velocity  $V$  is measured  $\pm 2\%$  as the average over the side of the field of view. The texture (bubble size, elongation and packing) and its variation (bubble stretching and rearrangements) enable to measure *in situ* the velocity gradient (not shown), the elastic deformation  $\pm 2\%$  and the plastic deformation rate  $\pm 7\%$  [19]. We plot here deviatoric terms, see section 3.3.

### 3.2 Resolution

Eqs. (1)-(5) can be solved in 2D or in 3D. Their main difficulties are intrinsic to VEP flows, independently of the details of the model. They reduce to a set of three partial differential equations with three unknowns ( $\varepsilon^e, \mathbf{v}, p$ ) and

the coupled system is highly non-linear : its numerical resolution needs to be performed carefully.

Here we solve these equations in 2D with a finite element algorithm first used for a simple Couette circular geometry [35], extended here to handle more complex flow domains [34]. As in 1D, the stationary solution is obtained by solving the time-dependent problem with a second order time-splitting algorithm, already used for VE [40], generalized here to VEP; it allows to treat separately the two main non-linearities of the equations, namely the plasticity term in eq. (3) and the stress transport term in the objective derivative (eq. (8)). Unlike in 1D [41], the stress transport term needs to be treated specifically by upwinding techniques; we chose a robust method, the discontinuous Galerkin scheme [42]. In addition, the non-linearity linked to plasticity needs a much more careful discretization than for the Couette resolution [35] to ensure a proper decreasing of the residue of the stationary problem. The spatial discretization is performed with a mixed finite-element method as in [30]. In order to get a general method suitable for any geometry, the domain is discretized with triangles.

The calculation domain is a channel,  $15 R = 22,5$  cm upstream and  $30 R = 45$  cm downstream of the obstacle. The mesh, made of 1100 triangles, is locally refined near the obstacle (see Fig. 5a). We start from a foam at rest and enforce the entrance velocity  $V$ . Unlike for most liquid flows, but in agreement with foam flow experiments [14], we use slip boundary conditions. Careful tests have been performed [34] in order to ensure that the mesh is suffi-

ciently refined and that discretization does not affect the results presented here. Iterations are performed (Fig. 5b) until the residue of the stationary problem is less than  $10^{-7}$  (see Fig. 5c). Calculations in 2D run in half a day on a Intel T7300 Core 2 Duo processor (2 GHz, 4 Mb cache, 32 bits). The 2D algorithm has been validated by reproducing the 1D Couette calculation [35], which runs in a few minutes.

### 3.3 Choice of parameters

The parameter with the main effect is  $\varepsilon_Y$ : a change of  $\pm 20\%$  suffices to visibly affects the main features of the flow. In a foam, its value is expected to depend mostly on the liquid fraction and on the dimension (2D or 3D), and possibly on the area dispersity; on the opposite, the average area and the physico-chemical properties of the foam are not expected to affect the value of  $\varepsilon_Y$ .

We have tested the effect of varying the other parameters, one by one. Like in Couette flow [35], they barely affect the flow, even if varied over a large range, as we now discuss.

The value  $k$  of the friction on the plates can be determined experimentally from the overall pressure gradient  $\nabla p$  across the channel length:  $k = \nabla p / V = 9300 \text{ Pa s m}^{-2}$ . If we take this value, we obtain a prediction indiscernable from  $k = 0$ . Only by taking a ten times larger friction,  $k = 10^5 \text{ Pa s m}^{-2}$ , does the change become visible. It means that here the limit  $k \rightarrow 0$  is regular. This case is similar to cylindrical Couette geometry, but different from planar Couette geometry [35].



We choose  $\eta_1/\eta_2 = 0.1$  according to the Couette case [35], since it lies in the middle of a range where its exact value barely affects the flow, even up to a factor of 10. Since the Reynolds number  $Re = \rho V R / (\eta_1 + \eta_2) \approx 5 \times 10^{-3}$ , we neglect the inertia term  $\rho \dot{\mathbf{v}}$  in Eq. (4).

We choose the co-rotational derivative [38], with  $a = 0$ , so that  $\varepsilon^e$  is deviatoric. In that case, the term  $\beta_a$  in eqs. (8-9) has a zero trace: hence if the trace of  $\varepsilon^e$  is initially zero, it remains zero [34]. In the low velocity regime explored here, any other choice of the objective derivative, with a parameter ranging from  $a = -1$ , lower-convected derivative, to  $a = 1$ , upper convected [38], would only change the trace of  $\varepsilon^e$ . Further investigations might determine the value of  $a$  by comparison with the experimental values of this trace.

The relaxation time  $\lambda$  is related to the Weissenberg number  $We = \lambda V / R$ . We distinguish three velocity regimes. At *high velocity*,  $We$  of order 1 or higher, the material rheology can display non-linearities in addition to those already present in the model. Since foams rupture at high velocities [39,43], this regime would be easier to investigate with other materials. In the *low velocity* range,  $We$  greater than  $10^{-2}$  but smaller than 1, the exact value of  $We$  does not affect the flow. This is the case for both experiments considered here, as well as for several foam flow experiments reported in the literature. Further decreasing  $We$  over two or three decades would lead to the *ultra slow* range, where the fore-aft asymmetry strongly increases. This is done in very few well-controlled experiments [39]. The limit  $We \rightarrow 0$  at constant  $\varepsilon_Y$  is singular: it implies a

divergence of the Bingham number  $Bi = 2\varepsilon_Y / We$  [11]. In fact, the ultra slow regime is not quasi-static [25,27,44] and does not exactly match quasi-static simulations [27].

### 3.4 Representations

Results are plotted either as maps or graphs.

Maps enable a global quantitative comparison between calculations and experiments: spatial distribution, magnitude, orientation of different quantities. Velocities are displayed as arrows. For clarity, they are presented in the average referential of the foam, as if a movable obstacle was displaced towards the left within a fixed foam, *i.e.*  $(v_x - V)/V$ . Traceless tensors are represented by circles. Their two eigenvalues have same absolute value. We indicate the direction of the positive one by a thin line. The direction of the negative one, not represented, is perpendicular. The top half, in red, is the present calculation using a continuous model, performed in the half-plane assuming top-bottom symmetry. Thin lines are isovalues of the stream function. The bottom half, in blue, are experimental data obtained as averages over bubbles. Thin lines are trajectories deduced from the experimental velocity field, plotted with approximately the same density as in calculations.

Graphs of components along selected axes enable detailed quantitative comparisons between calculations and experiments: position and amplitude of extrema, shape of curves, concavity, inflexion points, precision of the agreement. Axes are chosen to coincide with the measurements of ref. [14]: axis 1 is the symmetry axis, passing through

the obstacle center, parallel to the flow,  $y = 0$ ; axis 2 is parallel to the flow, close to the obstacle  $y = \pm 2.5$  cm; axis 3 is perpendicular to the flow, passing through the obstacle center,  $x = 0$ ; axis 4 is perpendicular to the flow, upstream of the obstacle,  $x = -2.4$  cm; axis 5 is perpendicular to the flow, downstream of the obstacle,  $x = 2.4$  cm. We do not plot components which vanish due to symmetry, such as  $v_y$  or  $\varepsilon_{xy}^e$  along the symmetry axis. Red lines are predictions, blue symbols are experimental tests.

## 4 Results

### 4.1 Wet foam flow

First, as a preliminary characterization, we study the flow of a wet foam (7% liquid fraction, Fig. 2a). We calculate the measurable fields: velocity  $\mathbf{v}$ , elastic deformation  $\varepsilon^e$ , plastic deformation rate  $\dot{\varepsilon}^p$ . We use the parameters obtained in ref. [35] and rescale them to the geometry of the present set-up. We investigate separately the effect of each parameter (see Section 3.3). We check that, with a yield strain  $\varepsilon_Y = 0.1 \pm 0.02$ , the calculations agree well simultaneously with all available experimental data (Fig. 3 and Figs. 6-8). Such value of  $\varepsilon_Y$  is the expected order of magnitude for a foam with this liquid fraction [19].

Other parameters have less effect. We use  $\lambda = 0.2 \pm 0.1$  s [14] and  $\eta_1/\eta_2 = 0.1$ . We take  $k = 0$  without significantly affecting the results. With a shear modulus estimated [14] around  $\mu = 13 \pm 1$  N.m<sup>-2</sup>, this translates into actual values:  $\sigma_Y = 2\mu\varepsilon_Y = 2.6$  N.m<sup>-2</sup>,  $\eta_2 = \lambda\mu = 2.6$  Pa.s and  $\eta_1 = 0.1 \eta_2 = 0.26$  Pa.s.

In such a VEP flow, elastic deformations are present in the wake of the obstacle at arbitrarily low velocity. Plasticity prevents the increase of extensional deformation, breaking the up-downstream (fore-aft) symmetry (Fig. 3). A velocity overshoot, the so-called “negative wake”, is clearly visible behind the obstacle (Fig. 3). This characteristic feature of VEP flows is barely affected even if we vary  $V$  across the low velocity regime, confirming experimental observations [14]. For instance, dividing  $V$  by 20 barely changes the overshoot (Fig. 3).

This strongly contrasts with VP flows, which are always fore-aft symmetric [30]. VE flows represent a mixed situation, where the negative wake has already been evidenced, both experimentally [45] and numerically [46, 47]. In fact, it occurs for low extensional viscosity fluids and models (*e.g.* FENE-CR [47] but not Oldroyd-B [31]), at elongational rates large enough in comparison with the inverse relaxation time, so that the elastic deformation does not vanish. However, at the low velocity investigated here, the VE flow is completely fore-aft symmetric (Fig. 3), and even indiscernable from viscous flows, whatever the viscosity. To interpret this set of observations, it seems that the overshoot appears when the elastic deformation ceases to follow passively the total deformation rate. This can occur if there is a mechanism which saturates the value of elastic deformation, which is the case in some VE models at high velocity, and in any VEP model because of plasticity.

To summarize this first test, by adjusting only one parameter, which value has the expected order of magnitude, we can adjust both qualitative and quantitative features

of all available data. We reproduce the observed negative wake and evidence the specificity of VEP flows.

## 4.2 Dry foam flow

Second, we turn to prediction. Since the parameter which has the most significant effect on the flow is the yield strain [14] (section 3.3), we choose to predict the flow for a twice larger value,  $\varepsilon_Y = 0.2$ .

These predictions are plotted on Fig. 4a, top and Fig. 4b-e, lines. The overshoot on the symmetry axis (Fig. 4b) is larger and closer to the obstacle than for  $\varepsilon_Y = 0.1$ : this reflects a larger effect of the elastic deformation. The elastic deformation field extends more than the velocity gradient (Fig. 4a, top), which itself extends more than the plastic deformation rate field (Fig. 11, top): this confirms that the three fields are physically independent quantities [19].

To check this prediction, we then perform a dry foam experiment, since decreasing the liquid fraction of a foam is expected to affect especially the yield strain. With 1.2% liquid fraction, Fig. 2b), we observe that  $\varepsilon_{max}^e$ , the maximum value of  $\varepsilon^e$  measured on the experiment, is twice as much that of the wet foam. The effect of elasticity is even stronger and the agreement with our prediction even better, without adjusting any parameter. Measurements confirm the predicted spatial distribution, magnitude, direction, anisotropy of fields.

The bubble *velocity* (Fig. 4) passes the three most stringent tests. First, the position and magnitude of overshoot on the symmetry axis (Fig. 4b). Second, the graph along the axis 5 (Fig. 4c). And third, the exact position of

the arrest points, defined in the referential of the foam, as points where  $v_x - V = 0$ : close to axis 5, on  $y = 0$  and  $\pm 5$  cm, see Fig. 4a. Other axes confirm the agreement (Fig. 9).

The bubble *elastic deformation* too agrees remarkably well in spatial distribution, amplitude and direction (Figs. 4a,d,e and Fig. 10). Interestingly, its orientation does not directly correlate with that of streamlines, or equivalently of the velocity vectors. This confirms that the elastic deformation should be treated as a variable independent of the velocity; we have also checked (data not shown) that it does not directly correlate with the total deformation rate. Note that residual normal stresses are visible beyond the obstacle. Unlike in the Couette case, here they are reproducible and their origin is understood: they are a direct effect of the obstacle, and do not depend on the foam preparation method.

The *plastic deformation rate* is calculated as the total deformation rate minus the elastic deformation rate (eq. 2). Its predicted spatial distribution and directions agree with that of the experimental measurements, which represent the time-averaged orientation, frequency and anisotropy of the bubble rearrangements (Fig. 11).

## 5 Discussion and conclusion

To summarize, a continuous description of viscous, elastic, plastic material with physically meaningful parameters can reproduce and even successfully predict a tensorial, spatially developed flow of a disordered rearranging structure.

We analyse and interpret the effect of each parameter separately. We emphasize the dominant role of elasticity and thus identify the yield strain as the most important parameter. The flow does not reduce to VE or VP separately, so that we emphasize the specific complexity of VEP materials.

Our method opens the way to computing two- or three-dimensional flows under any type or amplitude of deformation. It applies to those depending on one space variable: for instance a flow through a channel [15,16], or during simultaneous squeezing and shearing [6]. It also applies to those depending on two space variables: for instance a flow through a hole in 2D [19,48], or in 3D with axisymmetry. At the expense of longer calculations, it can even apply to flows which depend on three space variables, for instance through a twisted or branching pipe.

The model has been kept as simple as possible. On one hand, if we suppress one or another of its ingredients, our algorithm still can solve it, and the resulting solution lacks some of the experimental features. On the other hand, it can be progressively enriched by incorporating additional non-linearities, for instance related with the physico-chemistry of foams. When the velocity or the rigidity of the surfactant layer increases, shear thinning effects [26] could be introduced by an extended version of the VEP model [49] based on an Herschel-Bulkley viscoplasticity [29] instead of a Bingham one, as observed in [17,50]. The friction on walls too scales non-linearly with high velocity [26]. Non-linear elasticity at large deformation, although seldom reached in foams [19], can too be

taken into account [9,24]. At low velocity, plasticity seems to appear progressively: some bubbles begin to rearrange below the yield strain [19].

More generally, the model can be adapted specifically to any given VEP material of known properties. The value of the parameters of eqs. (1)-(5) depends on the micro-structure, its disorder and its physico-chemical properties. This is where the present approach can be enriched by statistical models based on the micro-structure [7,15,16,20,21,24,44,51].

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## Appendix

### Convergence

Fig. 5 represents how the time-dependent calculations converge towards the stationary solution. The residual term decreases first rapidly, until  $n = 200$ . It then reaches a plateau from  $n = 200$  to 3000 which corresponds to the advection of initial defects. It eventually reaches an asymptotical regime with a constant slope of convergence. Iterations were stopped when the residual term reached  $10^{-7}$ .

### Complete set of graphs

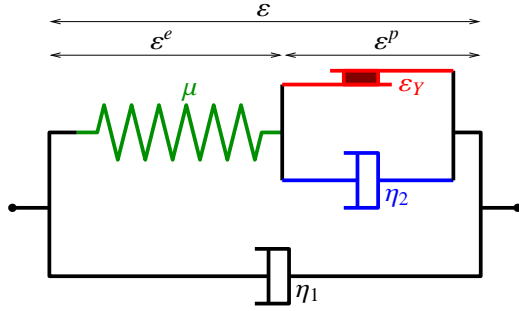
We have performed systematic calculations of all fields for several values of the parameters. They are all available upon request. Figs. 6-11 present the main ones.

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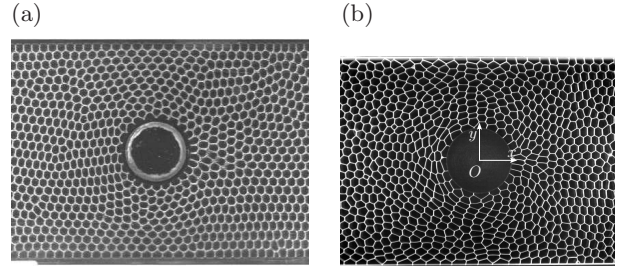
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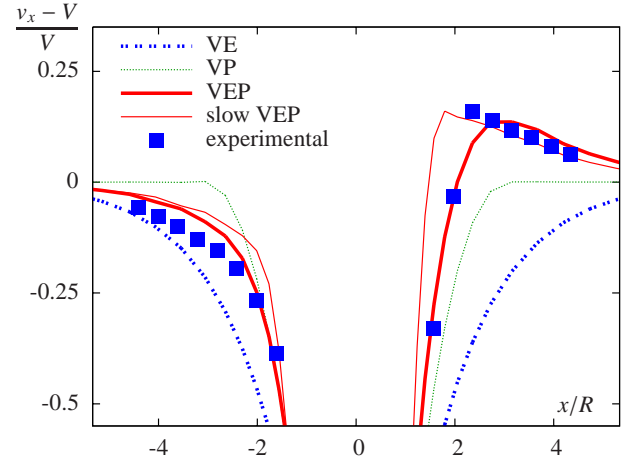
## Figures



**Fig. 1.** Symbolic representation of the viscous, elastic, plastic (VEP) model [11]: The spring symbolises an elastic stress proportional to the elastic deformation, and stands for the elastic modulus  $\mu$ . The dashpots, with a viscous stress proportional to the total deformation rate, stand for viscous dissipations  $\eta_1$ ,  $\eta_2$ . The solid friction element, which does not move until a sufficient force is applied, and exerts a constant resistance when it moves, stands for the plastic flow occurring when the elastic deformation is larger than the yield strain  $\varepsilon_Y$ .

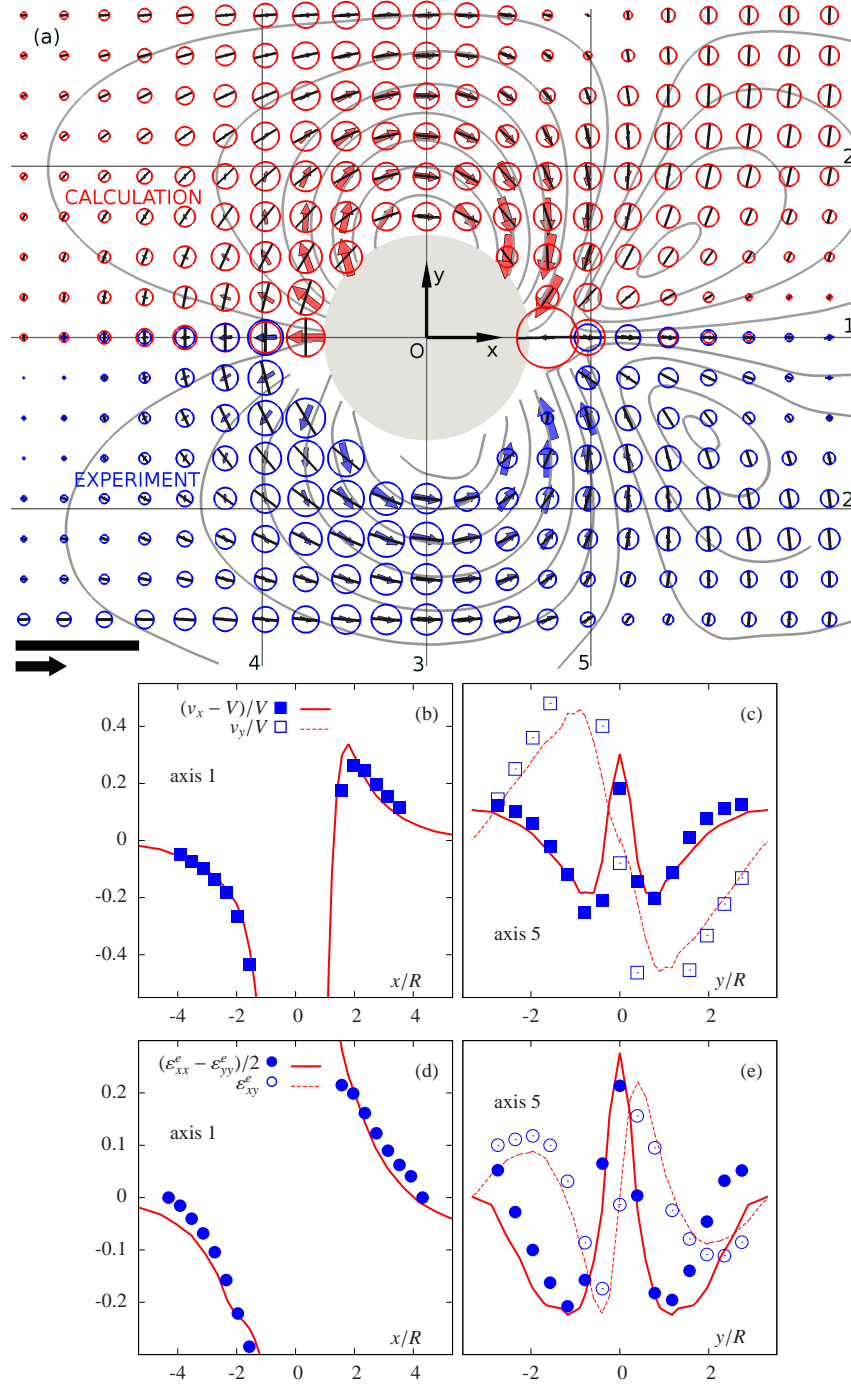


**Fig. 2.** Top view of the experiments. The foam is quasi-bidimensional, the flow (from left to right) is exactly bidimensional. The channel is rectangular: length 1 m (only partially shown), width 10 cm. The obstacle is circular, with radius  $R = 1.5$  cm. (a) Wet foam [14], liquid fraction  $\phi = 7\%$ , entrance velocity  $V = 1$  cm/s. (b) Dry foam,  $\phi = 1.2\%$ ,  $V = 0.6$  cm/s. Definition of axes  $x$  and  $y$ . The origin  $O$  is in the middle of the obstacle.



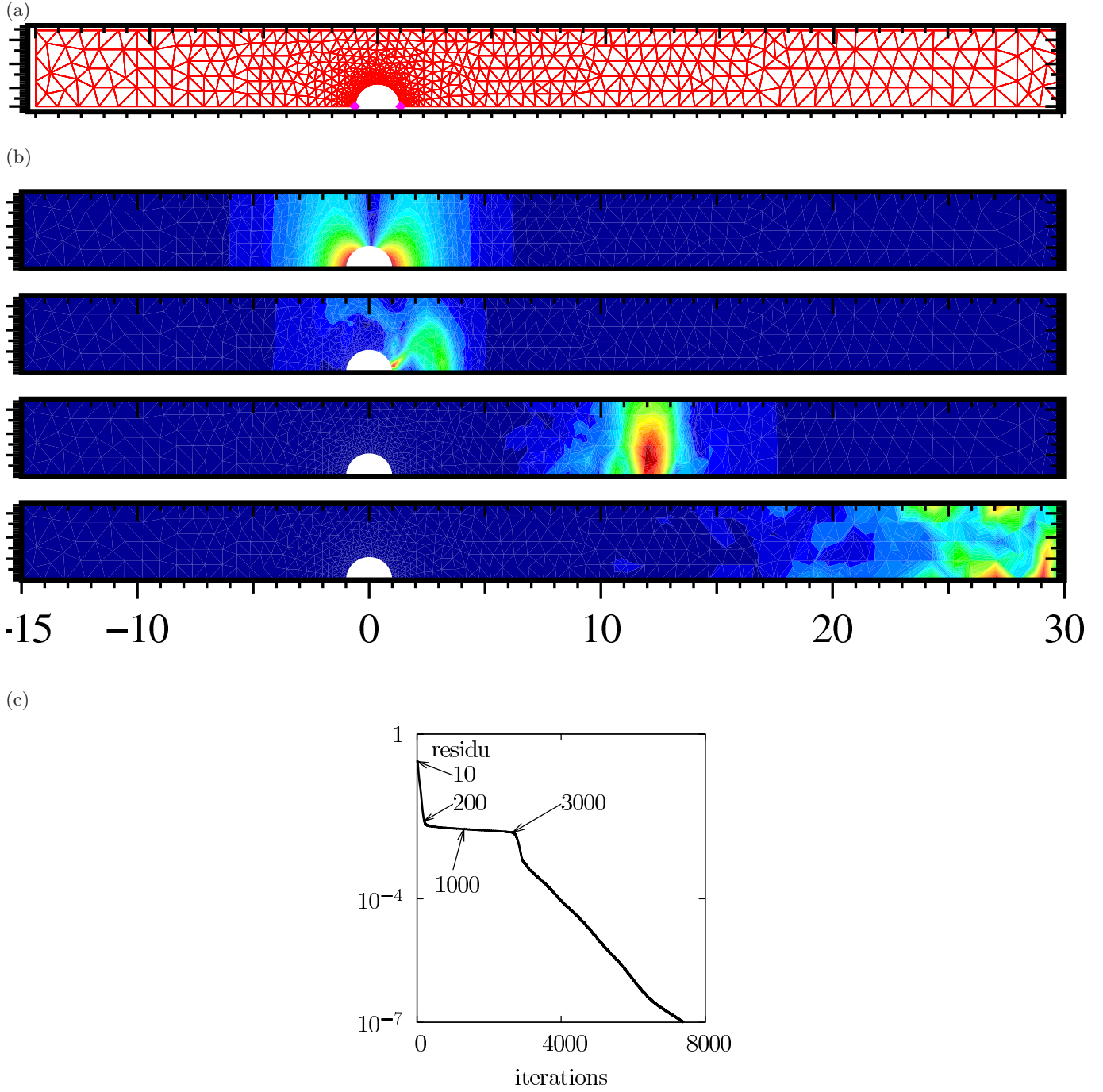
**Fig. 3.** Comparison of the flow around a circular obstacle with different models. Reduced velocity  $(v_x - V)/V$  along axis  $y = 0$  in wet foam experiment (closed squares, measured on Fig. 2a) is compared with the VEP calculation (present model,  $\varepsilon_Y = 0.1$ ,  $\lambda = 0.2$  s,  $\eta_1/\eta_2 = 0.1$ ,  $k = 0$ ), slow VEP (entrance velocity  $V$  divided by 20), VP (Bingham model [28], with the same Bingham number), VE (Oldroyd model [31], with the same  $\lambda$ ).



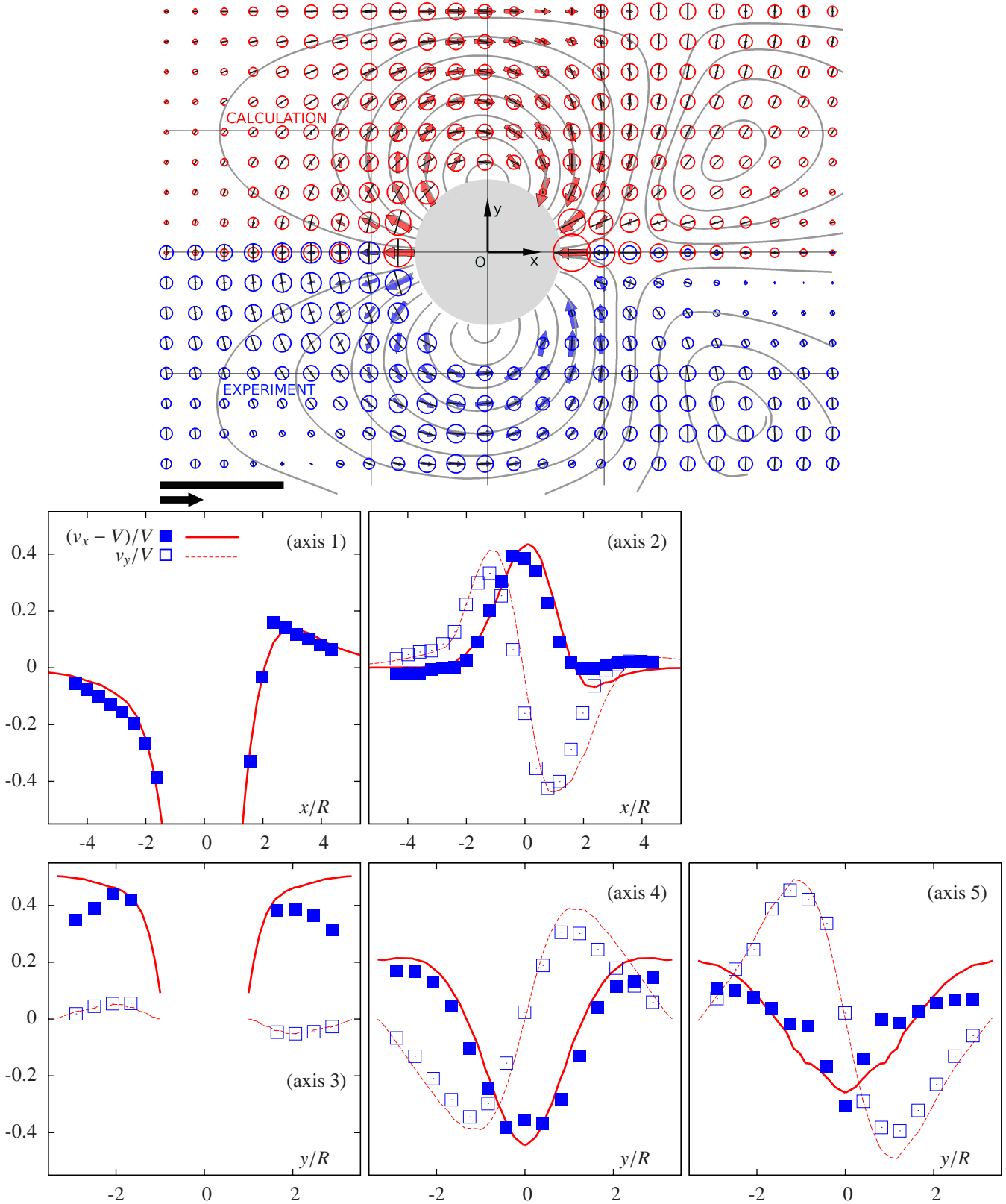


**Fig. 4.** Test of dry foam flow prediction, represented as explained in section 3.4. (a) Map. Thick colored arrows: velocity field. Circles: elastic deformation tensors; the positive (resp. negative) eigenvalue corresponding to elongation (resp. compression) is represented by a line (resp. not represented). Thin lines: stream lines. *Top half (red)*: present calculation using a continuous model.  $\varepsilon_Y = 0.2$ ,  $\lambda = 0.2$  s,  $\eta_1/\eta_2 = 0.1$ ,  $k = 0$ . *Bottom half (blue)*: experimental data obtained as averages over bubbles shown in Fig. 2b. Scale: bar: 1 (dimensionless) for the elastic eigenvalues (circle diameter); black arrow: entrance velocity  $V$ . (b-e) Graphs of the same data along axes 1 (b,d) and 5 (c,e). Red lines are predictions, blue symbols are experimental tests. (b,c):  $(v_x - V)/V$ : solid thick lines and closed squares;  $v_y/V$ : dashed thin lines and open squares. (d,e): elastic normal differences  $(\varepsilon_{xx}^e - \varepsilon_{yy}^e)/2$ : solid thick lines and closed circles; elastic shear  $\varepsilon_{xy}^e$ : dashed thin lines and open circles.

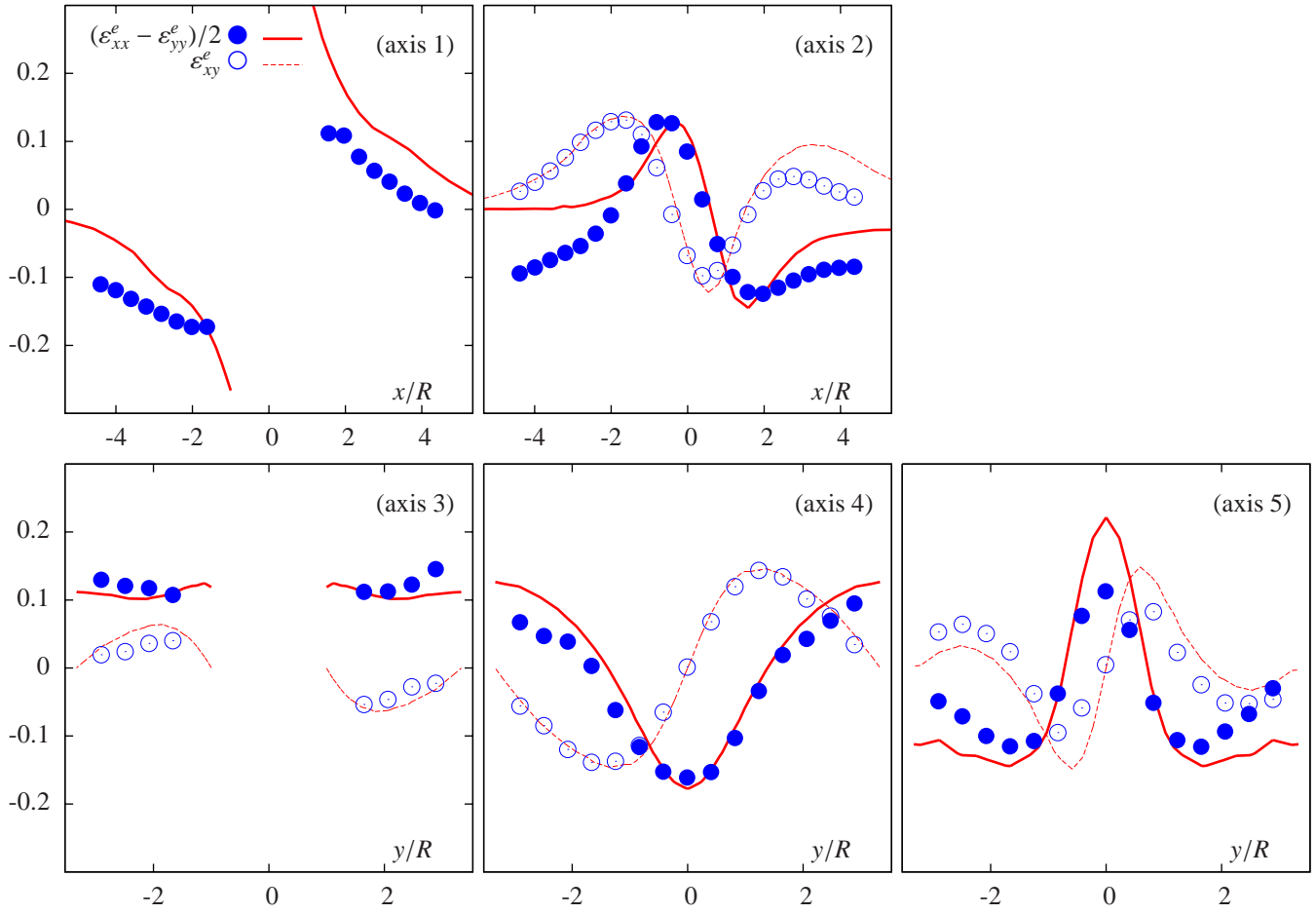




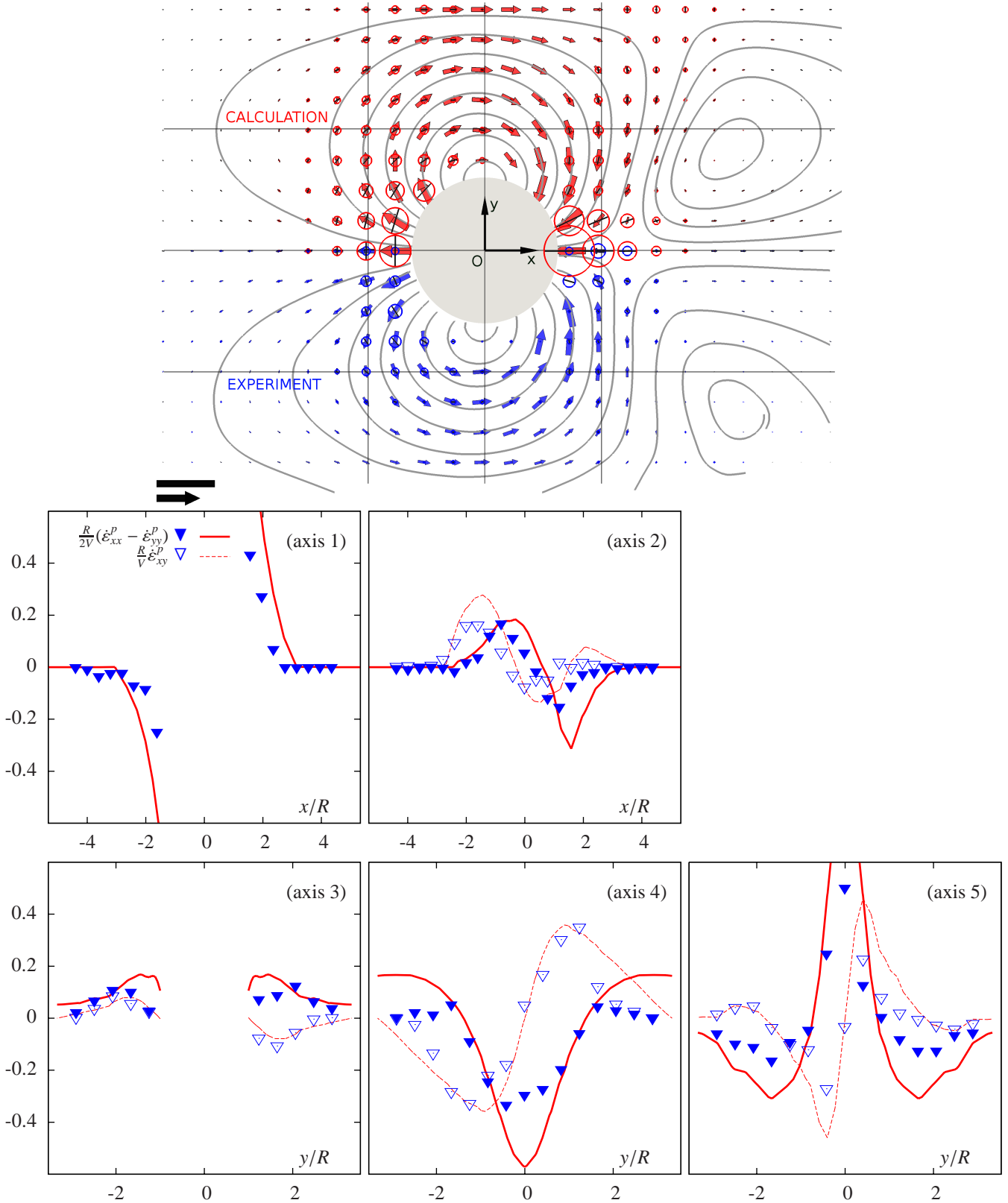
**Fig. 5.** Numerical convergence of calculations (see text). (a) Mesh used for the actual calculations; for convergence tests, the results have been compared with calculations performed on a finer mesh [34]. (b) The  $L^2$  norm of the stationary residues of eqs. (1-5) is displayed during the calculation which led to Figs. 4 and 9-11. From top to bottom:  $n = 10, 200, 1000$  and  $3000$  time iterations. Figures indicate  $x/R$ , where  $R$  is the obstacle radius. (c) The same  $L^2$  norm of the stationary residues is plotted versus time iterate.



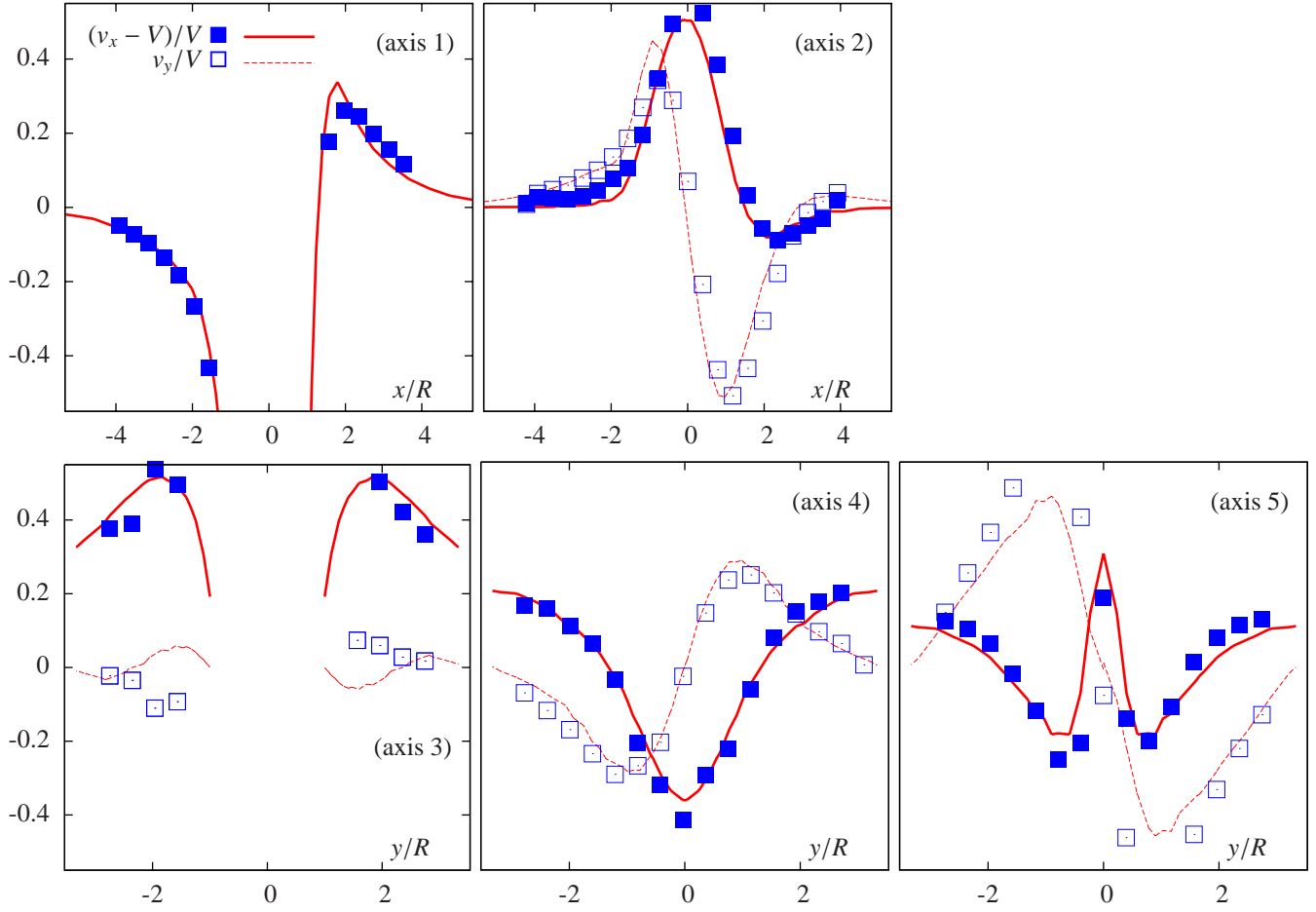
**Fig. 6.** Velocity and elastic deformation of wet foam ( $\varepsilon_Y = 0.1$ ). Red lines are calculations (same as Fig. 2a), blue symbols are experimental data (from Fig. 3). (top) Superimposition of velocity and elastic deformation fields, and streamlines. Scale: bar: 1 (dimensionless) for the elastic eigenvalues (circle diameter); black arrow: entrance velocity  $V$ . (bottom) Plots along axes 1-5. Same caption as Fig. 4.



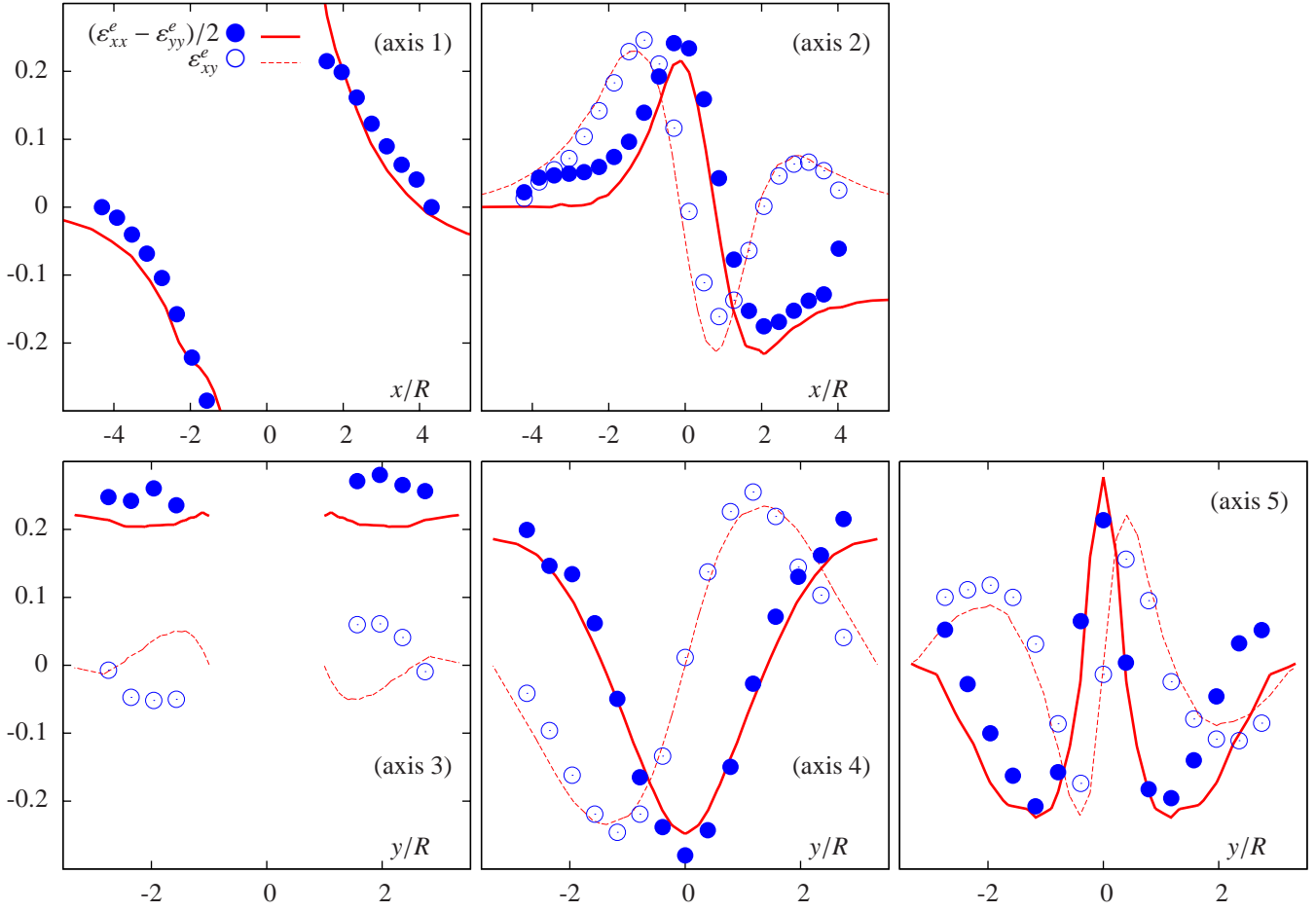
**Fig. 7.** Elastic deformation of wet foam (same as Figs. 2a and 3): plots along axes 1-5. Same caption as Fig. 4.



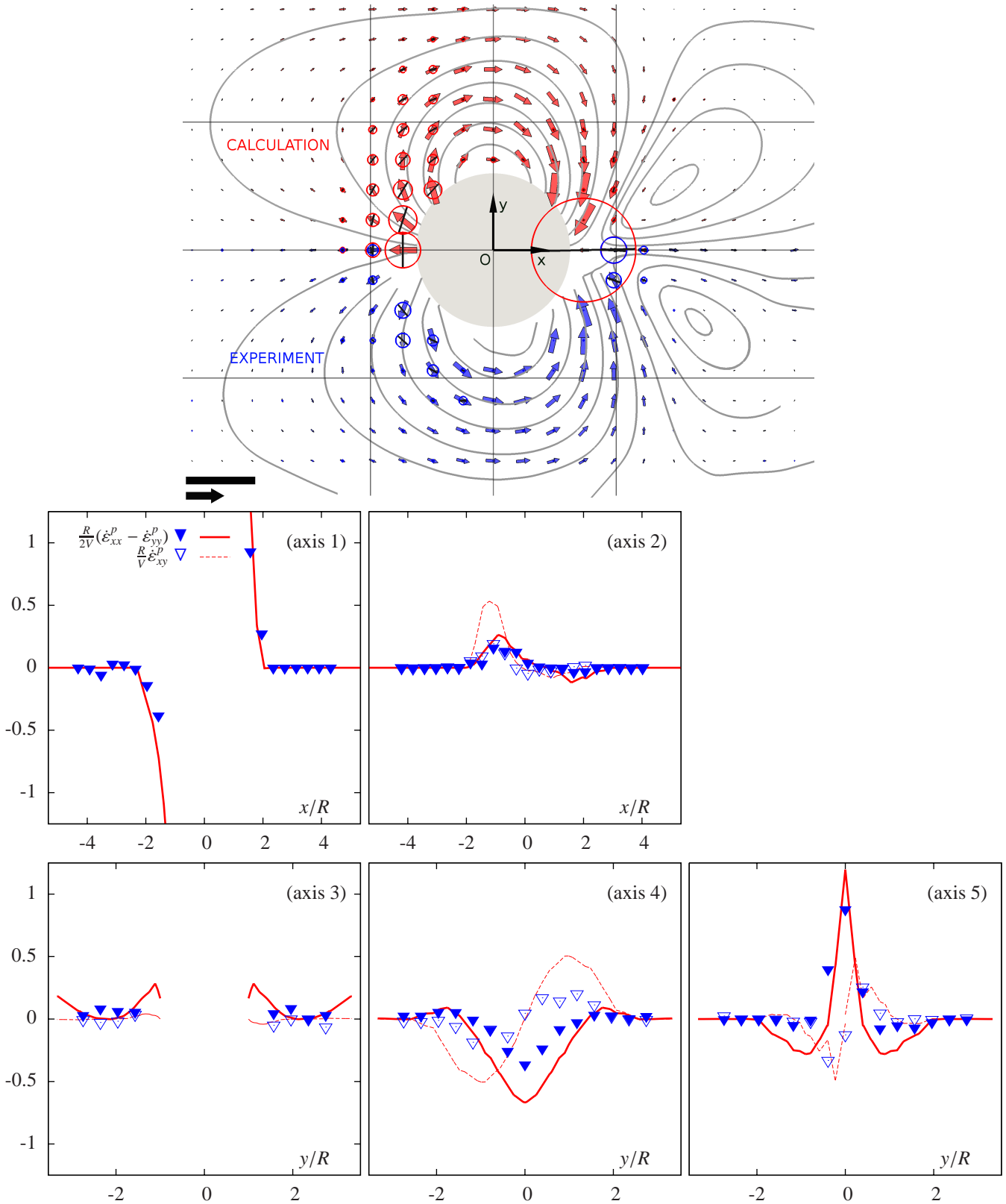
**Fig. 8.** Plastic deformation rate of wet foam, as Fig. 6. (top) Plastic deformation rate field, same caption as Fig. 4a. Scale: bar:  $1 \text{ s}^{-1}$  for the plastic deformation rate eigenvalues (circle diameter) ; black arrow: entrance velocity  $V$ . (bottom) Plots along axes 1-5. Red lines are calculations, blue symbols are experimental data:  $(\dot{\epsilon}_{xx}^p - \dot{\epsilon}_{yy}^p)R/2V$ : solid thick lines and closed triangles;  $\dot{\epsilon}_{xy}^p R/V$ : dashed thin lines and open triangles.



**Fig. 9.** Velocity of dry foam ( $\varepsilon_Y = 0.2$ ). Same caption and data as Fig. 4.



**Fig. 10.** Elastic deformation of dry foam. Same caption and data as Fig. 4.



**Fig. 11.** Plastic deformation rate of dry foam (same as in Figs. 2b and 4). (top) Plastic deformation rate field, same caption as Fig. 4. Scale: bar:  $1 \text{ s}^{-1}$  for the plastic deformation rate eigenvalues (circle diameter) ; black arrow: entrance velocity  $V$ . (bottom) Plots along axes 1-5. Same caption as Fig. 8.